

Newton's Law:

$$F = m a_{xyz}$$

①

A

Where the acceleration is measured relative to an inertial reference.

There are cases: — relative to noninertial reference in an airplane or rocket, where machine elements must move in a certain way relative to the vehicle in order to function properly. Therefore, the motion of the machine element relative to the vehicle is known.

if, however, the vehicle is undergoing a severe maneuver relative to inertial space, we can't use ①

∴

Multireference analysis. Attaching xyz to the vehicle and XYZ to inertial space

Then.

$$F = m [a_{xyz} + \ddot{R} + 2\omega \times V_{xyz} + \dot{\omega} \times R + \omega \times (\omega \times R)]$$

$$F - m(\ddot{R} + 2\omega \times V_{xyz} + \dot{\omega} \times R + \omega \times (\omega \times R)) = m a_{xyz}$$

Newton's Law in noninertial reference frame.

These terms $-m\ddot{R} - m(2\omega \times V_{xyz})$, $m(\dot{\omega} \times r)$, $m\omega \times (\omega \times r)$
are $\rightarrow$ forces $\rightarrow$ inertial forces. (2)

$-2m\omega \times V_{xyz} \rightarrow$ Coriolis forces

Inertial forces result in baffling actions that are sometimes contrary to our intuition.

⊗ Highly non-inertial $\rightarrow$ cases.

Fighter pilots and stunt pilots carry out actions in a cockpit of a plane while plane is undergoing severe maneuvers. Unexpected results frequently occur for flyer if they use the cockpit interior as reference for their actions.

Example 15.16

The plan view of a rotating platform is shown in Fig. 15.40. A man is seated at the position labeled A and is facing point O of the platform. He is carrying a mass of $\frac{1}{50}$ slug at the rate of 10 ft/sec in a direction straight ahead of him (i.e., toward the center of the platform). If this platform has an angular speed of 10 rad/sec and an angular acceleration of 5 rad/sec² relative to the ground at this instant, what force F must he exert to cause the mass to accelerate 5 ft/sec² toward the center?

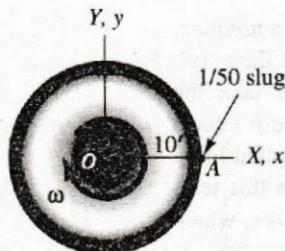


Figure 15.40. Rotating platform.

For purposes of determining inertial forces, we proceed as follows:

Fix xyz to platform.
Fix XYZ to ground.

Example 15.16 (Continued)

A. Motion of mass relative to xyz reference

$$\rho = 10i \text{ ft}, \quad V_{xyz} = -10i \text{ ft/sec}, \quad a_{xyz} = -5i \text{ ft/sec}^2$$

B. Motion of xyz relative to XYZ

$$\dot{R} = 0, \quad \ddot{R} = 0, \quad \omega = -10k \text{ rad/sec}, \quad \dot{\omega} = -5k \text{ rad/sec}^2$$

Hence,

$$a_{XYZ} = -5i + 2(-10k) \times (-10i) + (-5k) \times 10i + (-10k) \times (-10k \times 10i)$$

Therefore,

$$a_{XYZ} = -5i + (200j - 50j - 1,000i)$$

Employing Newton's law (Eq. 15.27) for the mass, we get

$$F - \frac{1}{50}(200j - 50j - 1,000i) = \frac{1}{50}(-5i)$$

Solving for F , we get

$$F = 3j - 20.1i \text{ lb}$$

This force F is the *total* external force on the mass. Since the man must exert this force and also withstand the pull of gravity (the weight) in the $-k$ direction, the force exerted by the man on the mass is

$$F_{\text{man}} = 3j - 20.1i + \frac{g}{50}k \text{ lb} \quad (a)$$

If the platform were *not* rotating at all, it could serve as an inertial reference. Then, we would have for the total external force F' :

$$F' = \frac{1}{50}(-5i) = -\frac{1}{10}i \text{ lb}$$

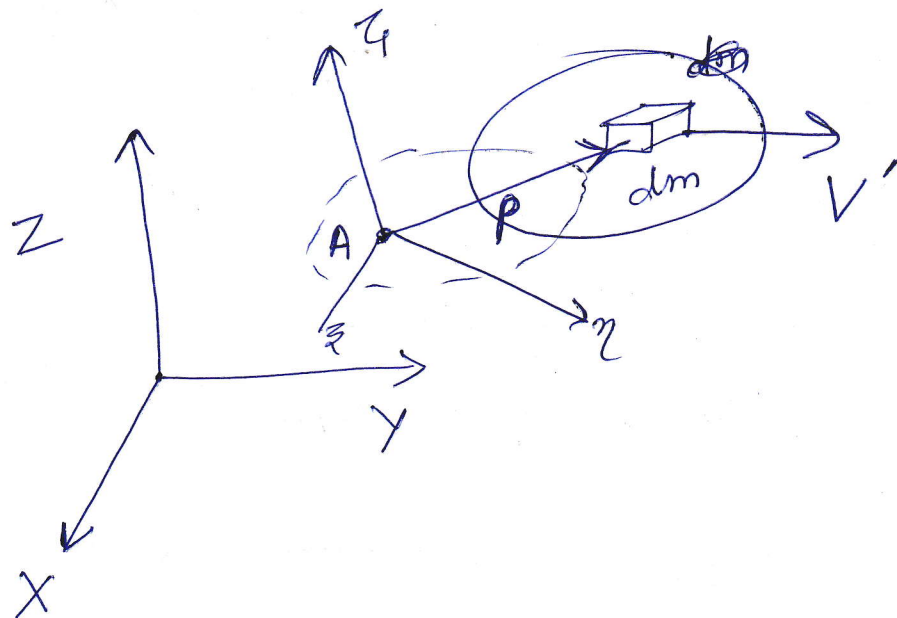
The force exerted by the man, F'_{man} , is then

$$F'_{\text{man}} = -\frac{1}{10}i + \frac{g}{50}k \text{ lb} \quad (b)$$

This force is considerably different from that given in Eq. (a).

As a matter of interest, we note that aviators of World War I were required to carry out such maneuvers on a rapidly rotating and accelerating platform so as to introduce them safely to these "peculiar" effects.

Dynamics of General Rigid body Motion.



Consider a rigid body moving arbitrarily relative to an inertial reference XYZ

Reference $\xi\eta\zeta$ translates with A.

A ~ point A in this body

dm = mass element of body at position P from A.

V' = Velocity of dm relative to A.
(Velocity of dm relative to reference $\xi\eta\zeta$ which translates with A relative to XYZ.)

Linear momentum of dm relative to A is the linear momentum of dm relative to $\xi\eta\zeta$ translating with A.

moment of this momentum (angular momentum) (2)

$$dH_A = \rho \times V' dm \Rightarrow \rho \times \left(\frac{d\rho}{dt} \right)_{xyz} dm$$

Now A is fixed in the body (or in the hypothetical massless extension of the body) then ρ must be fixed in the body, and, accordingly

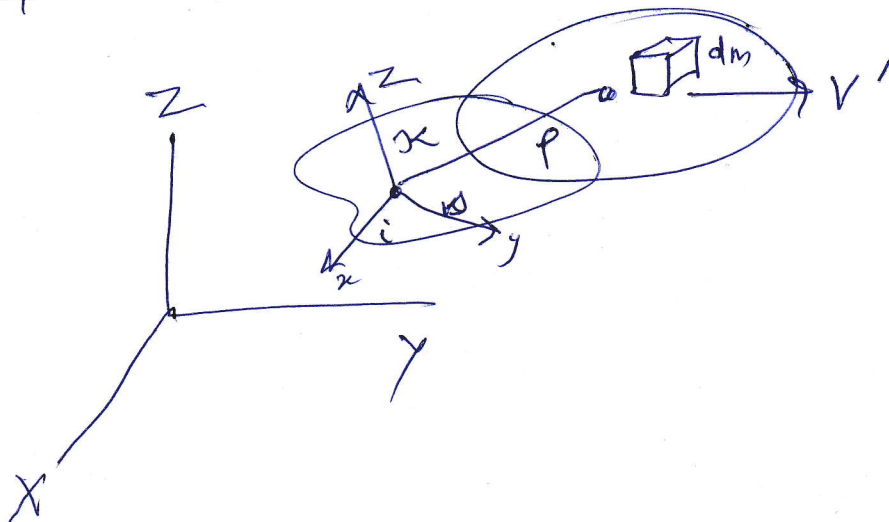
$$\left(\frac{d\rho}{dt} \right)_{xyz} = \omega \times \rho$$

ω = angular velocity of the body relative to xyz .

However, since xyz translates relative to XYZ, ω is the angular velocity of the body relative to XYZ as well.

$$dH_A = \rho \times (\omega \times \rho) dm$$

Now evaluate the components relative to xyz



(3)

$$(dH_A)_x \hat{i} + (dH_A)_y \hat{j} + (dH_A)_z \hat{k}$$

$$= \underbrace{(x\hat{i} + y\hat{j} + z\hat{k}) \times [\omega \times (x\hat{i} + y\hat{j} + z\hat{k})]}_{dm}$$

ω = angular velocity vector ~~w.r.t.~~ of body
w.r.t. XYZ.

Ω = angular velocity vector of reference xyz
w.r.t. XYZ.

$$\cancel{(dH_A)_x} = (x\hat{i} + y\hat{j} + z\hat{k}) \times \begin{bmatrix} \hat{i} & \hat{j} & \hat{k} \\ \omega_x & \omega_y & \omega_z \\ x & y & z \end{bmatrix} dm$$

$$= (x\hat{i} + y\hat{j} + z\hat{k}) \times \left[\begin{array}{l} \hat{i} \quad \hat{j} \quad \hat{k} \\ \hat{i} (z\omega_y - y\omega_z) - \hat{j} (\omega_x z - x\omega_z) \\ + \hat{k} (y\omega_x - x\omega_y) \end{array} \right] dm$$

$$= \begin{vmatrix} \hat{i} & \hat{j} & \hat{k} \\ x & y & z \\ z\omega_y - y\omega_z & x\omega_z - \omega_x z & y\omega_x - x\omega_y \end{vmatrix}$$

$$= \hat{i} [y^2\omega_x - yx\omega_y - zx\omega_z + z^2\omega_x] \\ - \hat{j} [xy\omega_x - x^2\omega_y - z^2\omega_y + zy\omega_z] \\ + \hat{k} [x^2\omega_z - xz\omega_x - yz\omega_y + y^2\omega_z]$$

Now

(4)

$$(dH_A)_x \hat{i} + (dH_A)_y \hat{j} + (dH_A)_z \hat{k} =$$

$$\begin{aligned} & \left[i \left[(y^2 + z^2) \omega_x - xy \omega_y - zx \omega_z \right] \right. \\ & + j \left[-xy \omega_x + (x^2 + z^2) \omega_y - zy \omega_z \right] \\ & \left. + k \left[-xz \omega_x - yz \omega_y + (x^2 + y^2) \omega_z \right] \right] dm \end{aligned}$$

Now Integrate all over the domain

$$\cancel{(H_A)_x \hat{i} + (H_A)_y \hat{j} + (H_A)_z \hat{k}}$$

$$\cancel{= \left[\omega_x \iiint_V (x^2 + y^2) dm - \omega_y \iiint_V xy dm - \omega_z \iiint_V zx dm \right] \hat{i}}$$

$$\cancel{+ \left[-\omega_x \iiint_V xy dm + \omega_y \iiint_V (x^2 + z^2) dm - \omega_z \iiint_V yz dm \right] \hat{j}}$$

$$(H_A)_x \hat{i} + (H_A)_y \hat{j} + (H_A)_z \hat{k}$$

$$= \begin{bmatrix} (I_{xx} \omega_x - I_{xy} \omega_y - I_{xz} \omega_z) \hat{i} \\ + (-I_{yx} \omega_x + I_{yy} \omega_y - I_{yz} \omega_z) \hat{j} \\ + (-I_{zx} \omega_x - I_{zy} \omega_y + I_{zz} \omega_z) \hat{k} \end{bmatrix}$$

(5)

$$\iiint_{dm} (y^2 + z^2) dm = I_{xx}$$

Now

$$(H_A)_x = I_{xx} \omega_x - I_{xy} \omega_y - I_{xz} \omega_z$$

$$(H_A)_y = -I_{yx} \omega_x + I_{yy} \omega_y - I_{yz} \omega_z$$

$$(H_A)_z = -I_{zx} \omega_x - I_{zy} \omega_y + I_{zz} \omega_z$$

Euler's Equation of Motion

Now We can arrange in terms of inertia matrix

$$\begin{bmatrix} I_{xx} & -I_{xy} & -I_{xz} \\ -I_{yx} & I_{yy} & I_{yz} \\ -I_{zx} & I_{zy} & I_{zz} \end{bmatrix}$$

It can be shown^v that there is one unique orientation of axes x-y-z for a given origin for which the product of inertia vanish and moment of inertia I_{xx} , I_{yy} , I_{zz} take on stationary values

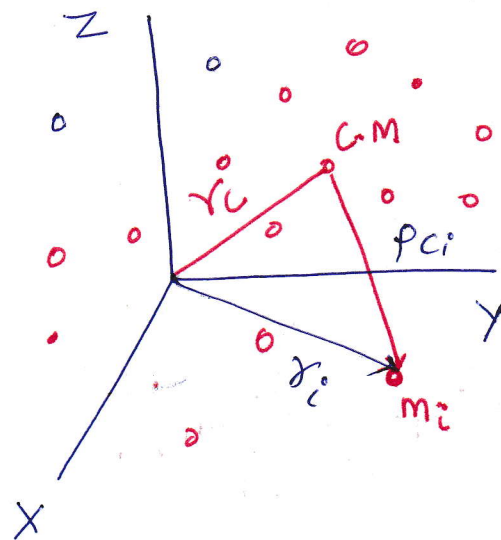
$$\begin{bmatrix} I_{xx} & 0 & 0 \\ 0 & I_{yy} & 0 \\ 0 & 0 & I_{zz} \end{bmatrix}$$

Kinetic Energy of a Rigid Body

Kinetic Energy of a particle

$$K.E = \frac{1}{2} m v^2 = \frac{1}{2} m \dot{x}^2$$

Kinetic Energy of a system of particles:



$$r_i = r_c + p_{ci} \quad \text{--- (1)}$$

Differentiating w.r.t. time.

$$\frac{dr_i}{dt} = \frac{dr_c}{dt} + \frac{dp_{ci}}{dt}$$

$$V_i = V_c + \dot{p}_{ci}$$

Where $\dot{p}_{ci} \Rightarrow$ motion of the i th particle relative to mass center.

$$K.E = \sum_{i=1}^n \frac{1}{2} m_i (V_c + \dot{p}_{ci})^2 = \sum_{i=1}^n \frac{1}{2} m_i (V_c + \dot{p}_{ci}) \cdot (V_c + \dot{p}_{ci})$$

$$K.E = \frac{1}{2} \sum_{i=1}^n m_i v_c^2 + \sum_{i=1}^n m_i v_c \cdot \dot{r}_{ci} + \frac{1}{2} \sum_{i=1}^n m_i \dot{r}_{ci}^2$$

Now

$$K.E = \frac{1}{2} \sum_{i=1}^n m_i v_c^2 + v_c \cdot \sum_{i=1}^n m_i \dot{r}_{ci} + \frac{1}{2} \sum_{i=1}^n m_i \dot{r}_{ci}^2$$

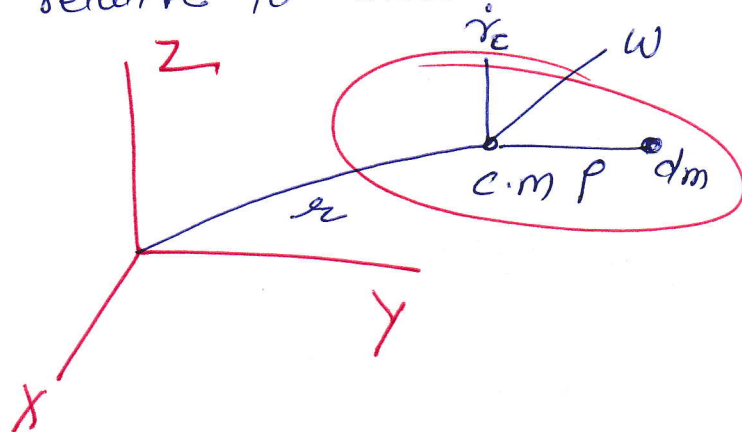
$$v_c \cdot \frac{d}{dt} \sum_{i=1}^n (m_i r_{ci})$$

$$\sum_{i=1}^n m_i r_{ci} = 0$$

— first moment of mass of the system about the center of mass.

$$K.E = \frac{1}{2} M v_c^2 + \frac{1}{2} \sum_{i=1}^n m_i \dot{r}_{ci}^2$$

Let us consider the above equation applied to rigid body. In such case, the velocity of any particle relative to mass center becomes —

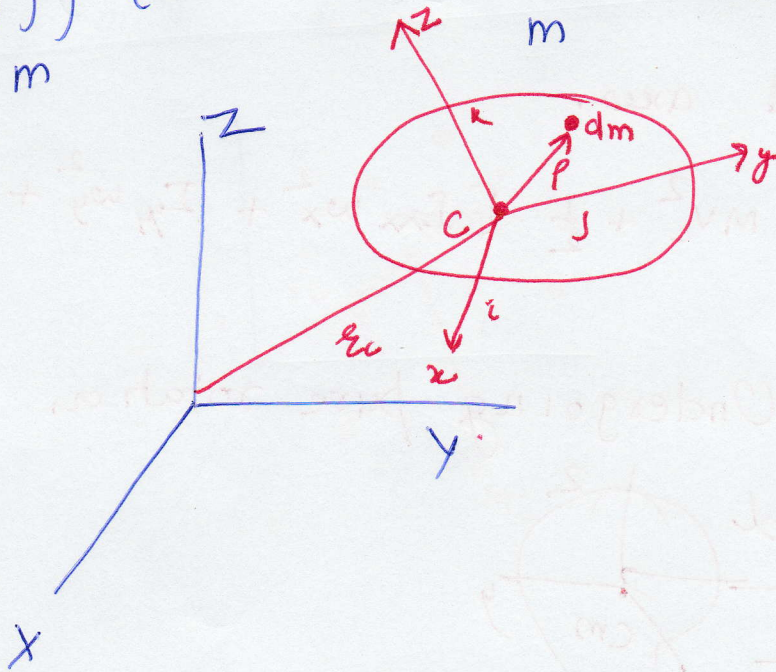


$$\dot{r}_i = \underline{\underline{\omega \times r_i}}$$

$$K.E = \frac{1}{2} M v_c^2 + \frac{1}{2} \int \int \int_m (\omega \times r_i)^2 dm$$

Now choose a set of orthogonal directions XIII
 xyz at the center of mass, so we can
 carry out the preceding integration in terms
 of the scalar components ω and ρ .

$$\iiint_m (\omega \times \rho)^2 dm = \iiint_m (\omega \times \rho) \cdot (\omega \times \rho) dm$$



$$\iiint_m (\omega \times \rho)^2 dm = \iiint_m \left[(\omega_x \hat{i} + \omega_y \hat{j} + \omega_z \hat{k}) \times (x\hat{i} + y\hat{j} + z\hat{k}) \right] \cdot \left[(\omega_x \hat{i} + \omega_y \hat{j} + \omega_z \hat{k}) \times (x\hat{i} + y\hat{j} + z\hat{k}) \right] dm$$

Carrying out first cross-product and dot product
 in the integrand and collecting terms.

$$\begin{aligned} &= \left[\iiint_m (z^2 + y^2) dm \right] \omega_x^2 - \left[\iiint_m (xy) dm \right] \omega_x \omega_y - \left[\iiint_m (xz) dm \right] \omega_x \omega_z \\ &\quad - \left[\iiint_m (yz) dm \right] \omega_y \omega_z + \left[\iiint_m (x^2 + z^2) dm \right] \omega_y^2 - \left[\iiint_m (yz) dm \right] \omega_y \omega_z \\ &\quad - \left[\iiint_m (zx) dm \right] \omega_z \omega_x - \left[\iiint_m (zy) dm \right] \omega_z \omega_y + \left[\iiint_m (x^2 + y^2) dm \right] \omega_z^2 \end{aligned}$$

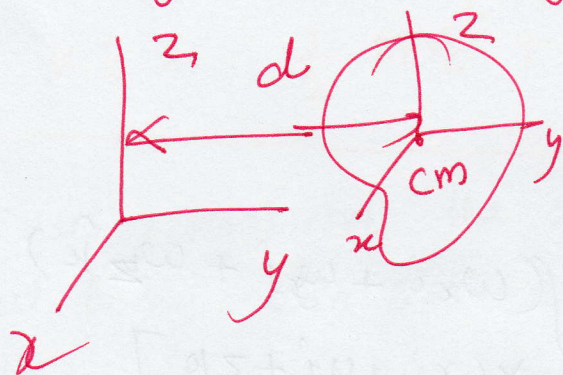
Integrals are the moments and product of inertia for the xyz references.

$$\iiint |\omega \times r|^2 dm = I_{xx} \omega_x^2 - I_{xy} \omega_x \omega_y - I_{xz} \omega_x \omega_z - I_{yx} \omega_x \omega_y + I_{yy} \omega_y^2 - I_{yz} \omega_y \omega_z - I_{zx} \omega_z \omega_x - I_{zy} \omega_z \omega_y + I_{zz} \omega_z^2$$

For Principal axes.

$$K.E = \frac{1}{2} M V_c^2 + \frac{1}{2} [I_{xx} \omega_x^2 + I_{yy} \omega_y^2 + I_{zz} \omega_z^2]$$

Rigid body Undergoing pure rotation



rigid body undergoing pure rotation relative to XYZ with angular velocity ω

$$\omega_x = \omega_y = 0, \quad \omega_z = \omega$$

$$K.E = \frac{1}{2} M V_c^2 + \frac{1}{2} I_{zz} \omega^2$$