

# 3-D Kinematics of Rigid Bodies

## Rotation about a Fixed Point

### Instantaneous Axis of Rotation

:: Rotation of solid rotor with black particles embedded on its surface

::  $\omega_1$  and  $\omega_2$  are steady angular velocities

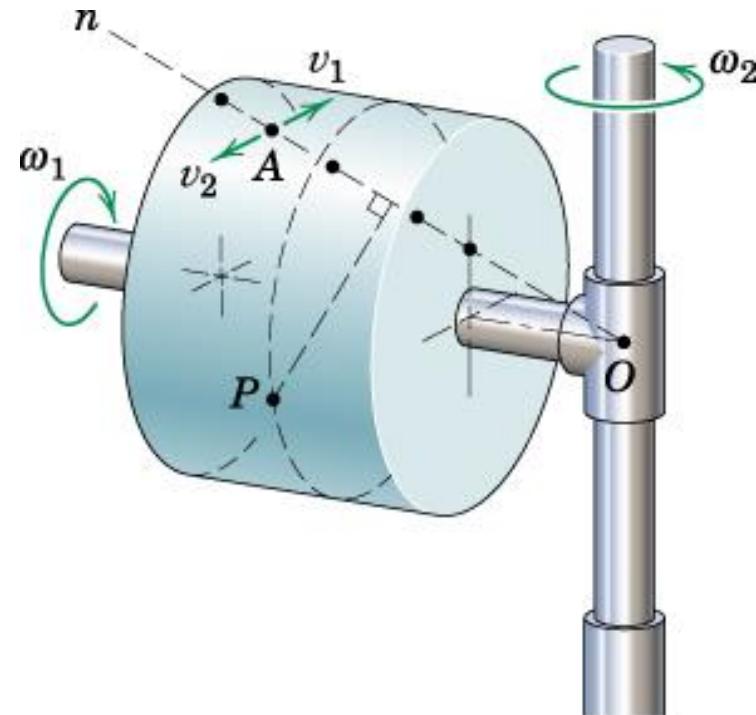
:: Along line  $O-n$ , velocity of particles will be momentarily zero (the line will be sharply defined)

:: The line of points with no velocity

→ Instantaneous position of the Axis of Rotation

:: Any point on this line, e.g.,  $A$ , would have equal and opposite velocity components  $v_1$  and  $v_2$  due to  $\omega_1$  and  $\omega_2$ , respectively

All particles of the body, except those on line  $O-n$ , rotate momentarily in circular arcs @ the instantaneous axis of rotation



# 3-D Kinematics

## Rotation about a Fixed Point

### Instantaneous Axis of Rotation

:: Change of position of rotation axis both in space and relative to the body

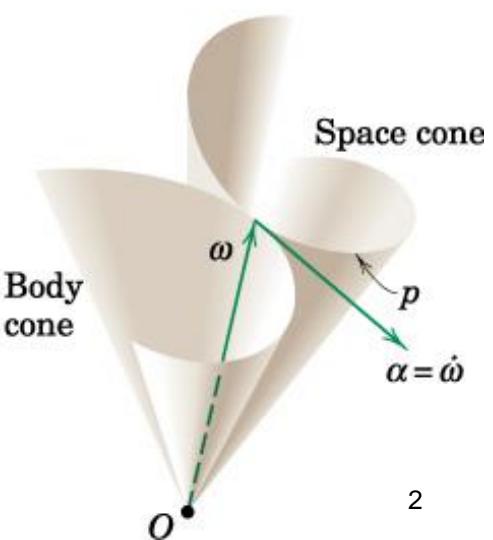
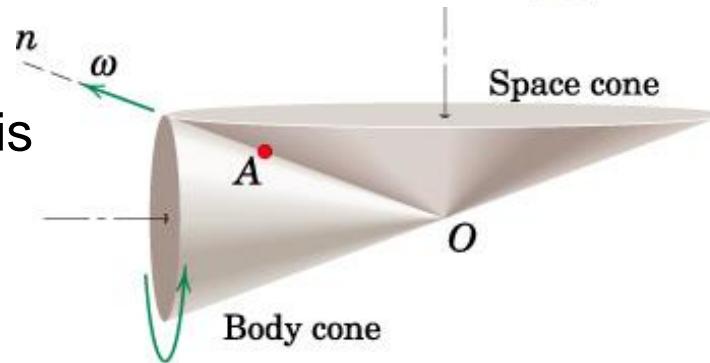
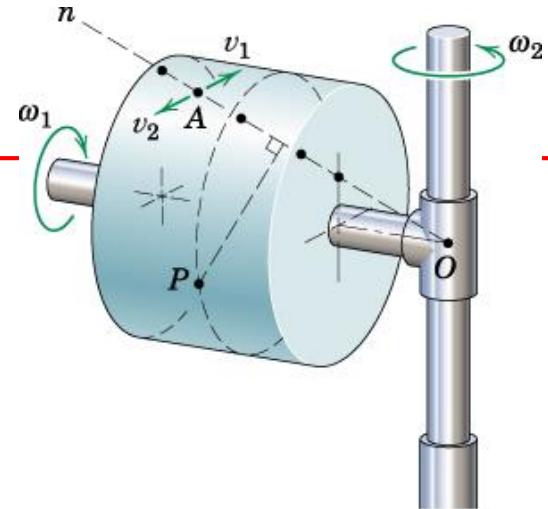
### Body and Space Cones

:: Relative to the cylinder, the instantaneous axis of rotation  $O-A-n$  generates a right circular cone @ the cylinder axis  $O \rightarrow \mathbf{Body\ Cone}$

:: Instantaneous axis of rotn also generates a right circular cone @ the vertical axis  $\rightarrow \mathbf{Space\ Cone}$

:: Rolling of body cone on space cone

:: Angular vel  $\omega$  of the body located along the common element of the two cones



# 3-D Kinematics

## Rotation about a Fixed Point

### Angular Acceleration

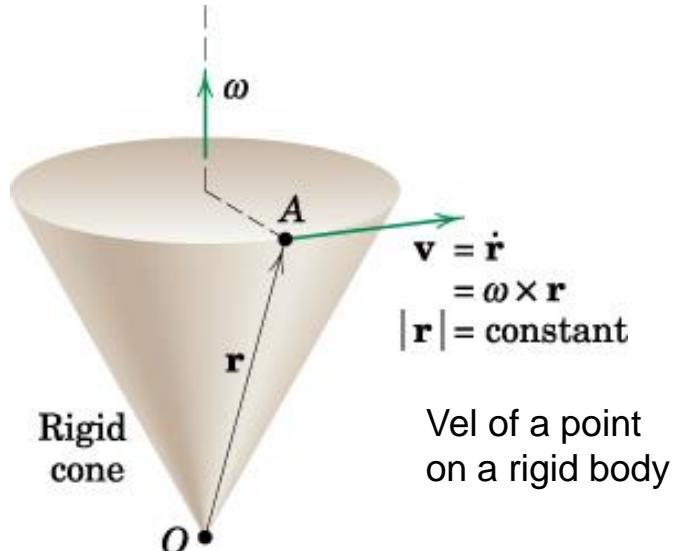
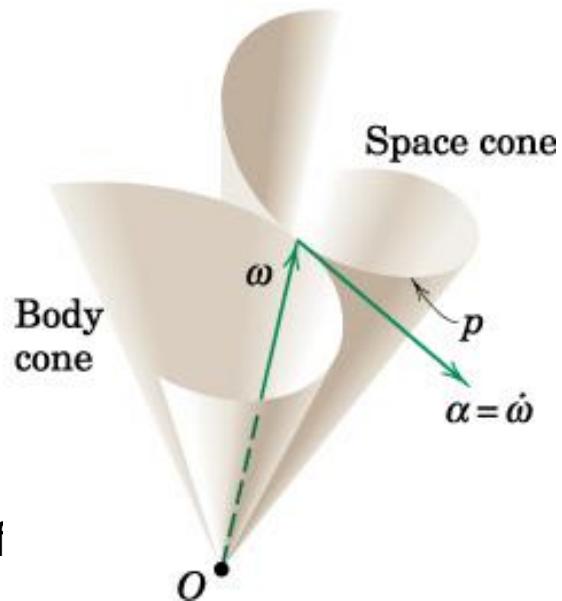
:: Planar motion

- scalar  $\alpha$  reflects the change in magnitude of  $\omega$

:: 3-D motion

- vector  $\alpha$  reflects the change in the direction and magnitude of  $\omega$

::  $\alpha$  - tangent vector to the space curve  $p$  in the direction of the change in  $\omega$



# 3-D Kinematics

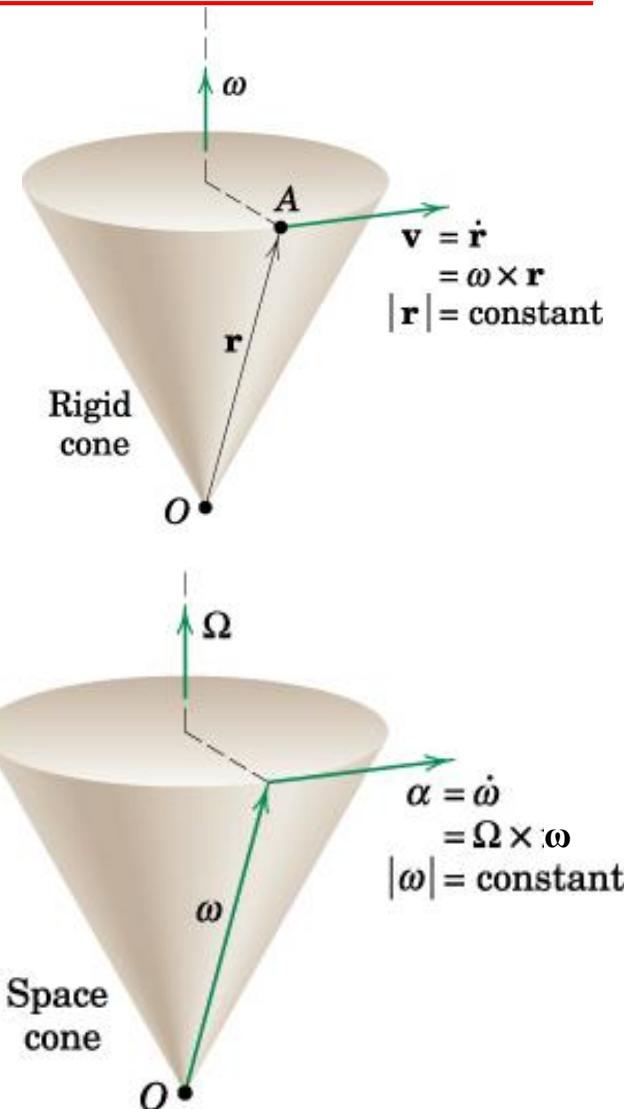
## Rotation about a Fixed Point

If the magnitude of  $\omega$  remains constant,  $\alpha$  is normal to  $\omega$

Let  $\Omega$  be the angular velocity with which the vector  $\omega$  itself rotates (precesses) as it forms the space cone

$$\rightarrow \alpha = \Omega \times \omega$$

$\therefore \alpha, \Omega$ , and  $\omega$  have the same relationship as that of  $v, \omega$ , and  $r$  defining the vel of a point A in a rigid body



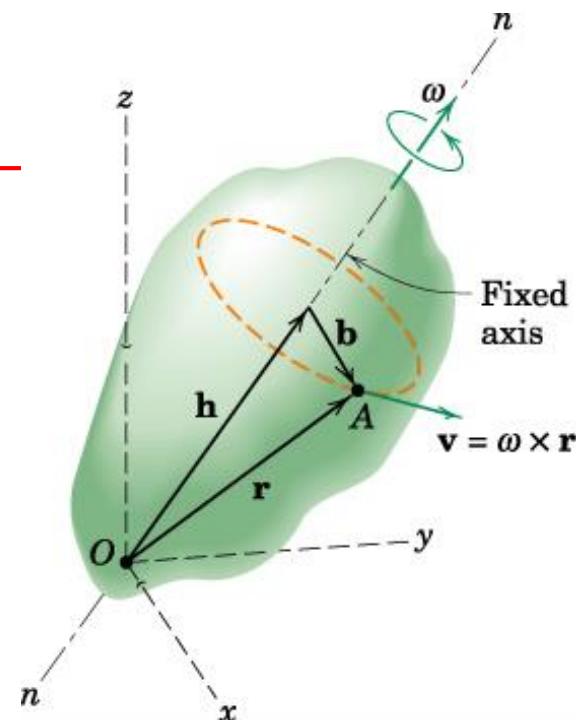
# 3-D Kinematics

## Rotation about a Fixed Point

When a rigid body rotates @ a fixed point O with the instantaneous axis of rotation  $n-n$ , vel  $\mathbf{v}$  and accln  $\mathbf{a} = \dot{\mathbf{v}}$  of any point A in the body are given by the same expressions derived when the axis was fixed:

$$\mathbf{v} = \boldsymbol{\omega} \times \mathbf{r}$$

$$\mathbf{a} = \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r})$$



## Rotation about a Fixed Axis vs Rotation about a Fixed Point

### For rotation @ a fixed point:

- angular accln will have a comp normal to  $\boldsymbol{\omega}$  due to change in dirn of  $\boldsymbol{\omega}$ , as well as a comp in the dirn of  $\boldsymbol{\omega}$  to reflect any change in the magnitude of  $\boldsymbol{\omega}$
- Although any point on the rotation axis  $n-n$  momentarily will have zero velocity, it will not have zero accln as long as  $\boldsymbol{\omega}$  is changing its direction

### For rotation @ a fixed axis:

- angular accln will have only one component along the fixed axis to reflect any change in the magnitude of  $\boldsymbol{\omega}$ .
- Points which lie on the fixed axis of rotation will have no velocity or accln

# 3-D Kinematics

## General Motion in Three Dimension

:: Principles of relative motion (both translating & rotating reference axes)

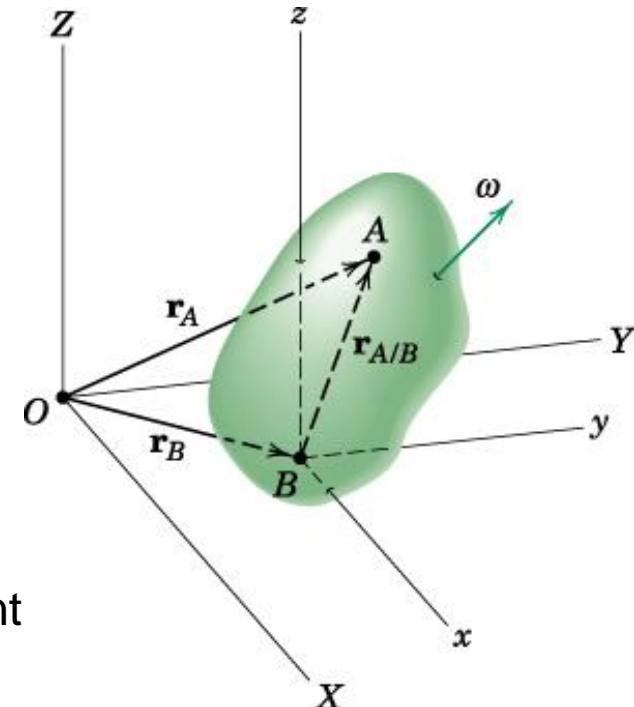
### Translating Reference Axes

- $X-Y-Z$  :: fixed reference frame
- $x-y-z$  :: translating reference frame with any point  $B$  as the origin
- $\omega$  :: angular velocity of the rigid body

Linear velocity and accln of any point  $A$  in the body

$$\mathbf{v}_A = \mathbf{v}_B + \mathbf{v}_{A/B} \quad \text{-3 coplanar vectors in each eqn}$$

$$\mathbf{a}_A = \mathbf{a}_B + \mathbf{a}_{A/B} \quad \text{- Distance } \overline{AB} \text{ should remain constant}$$



- To an observer on  $x-y-z$ , the body will appear to rotate @  $B$
- Point  $A$  will appear to lie on a spherical surface with  $B$  as the center

→ General Motion  $\equiv$  translation of body with motion of  $B$   
+ rotation of body @  $B$

# 3-D Kinematics :: General motion

## Translating Reference Axes

$$\mathbf{v}_A = \mathbf{v}_B + \mathbf{v}_{A/B}$$

$$\mathbf{a}_A = \mathbf{a}_B + \mathbf{a}_{A/B}$$

Relative motion terms represent effect of rotation about point  $B$

→ These are identical to the vel and accln relations obtained for a rigid body rotating @ a fixed point:

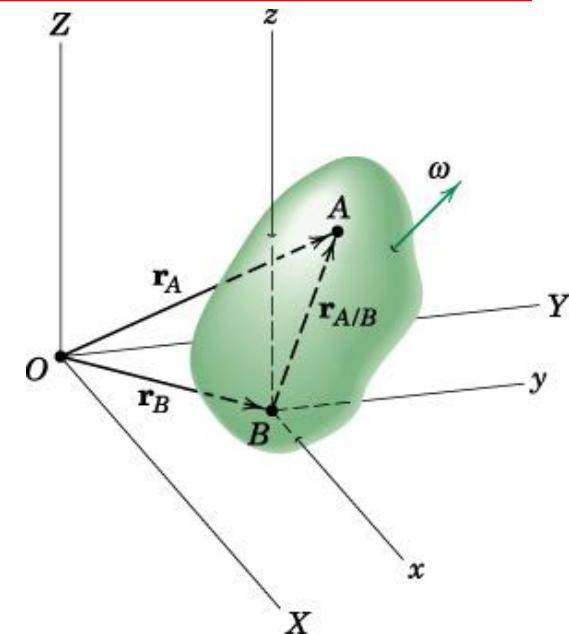
$$\mathbf{v} = \boldsymbol{\omega} \times \mathbf{r}$$

$$\mathbf{a} = \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r})$$

Relative terms

$$\rightarrow \mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega} \times \mathbf{r}_{A/B}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}} \times \mathbf{r}_{A/B} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}_{A/B})$$



$\boldsymbol{\omega}$  and  $\dot{\boldsymbol{\omega}}$  :: instantaneous angular vel and angular accln of the body

If point  $A$  is chosen as a reference point then vel and accln relations for point  $B$ :

$\mathbf{r}_{B/A} = -\mathbf{r}_{A/B}$ .  $\boldsymbol{\omega}$  &  $\dot{\boldsymbol{\omega}}$  will be same for both cases

$$\mathbf{v}_B = \mathbf{v}_A + \boldsymbol{\omega} \times \mathbf{r}_{B/A}$$

$$\mathbf{a}_B = \mathbf{a}_A + \dot{\boldsymbol{\omega}} \times \mathbf{r}_{B/A} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}_{B/A})$$

# Example (1) on general motion

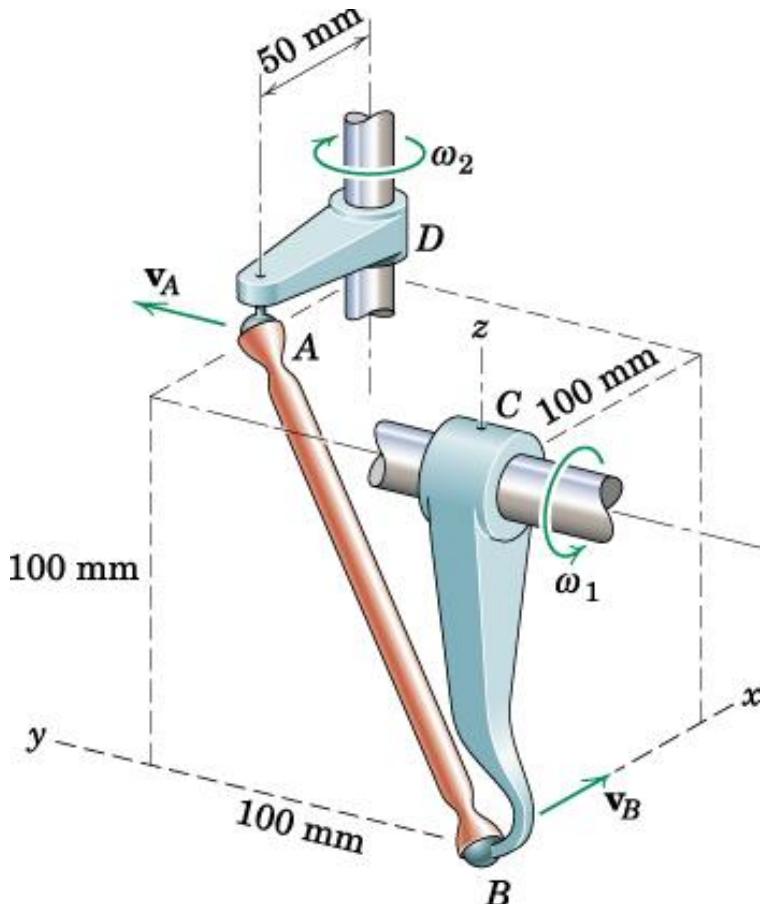
**Example:** Crank *CB* rotates @ the horz axis with an ang vel  $\omega_1 = 6 \text{ rad/s}$ , which is constant for a short interval of motion including the position shown. Link *AB* has a ball-and-socket fitting on each end and connects crank *DA* with *CB*. For the instant shown, det the ang vel  $\omega_2$  of crank *DA* and ang vel  $\omega_n$  of link *AB*. Further determine, the ang accln  $\dot{\omega}_2$  of crank *DA* and ang accln  $\dot{\omega}_n$  of link *AB*.

### Solution:

Choose point  $B$  as the origin of translating reference axes.

## Relative velocity relation:

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega}_n \times \mathbf{r}_{A/B}$$



# Example (1) on general motion

Relative velocity relation:

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega}_n \times \mathbf{r}_{A/B}$$

$\boldsymbol{\omega}_n$  is the angular vel of link  $AB$  taken normal to  $AB$

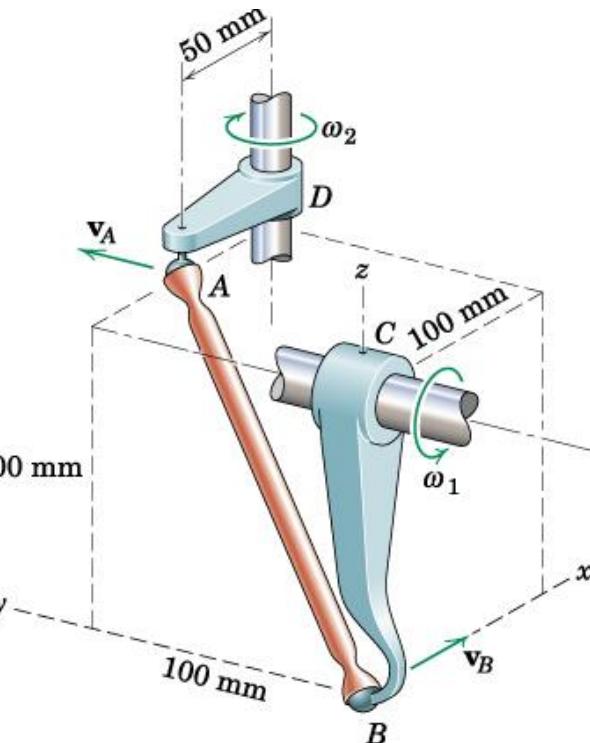
:: The angular vel of  $AB$  is taken as normal to  $AB$  since any rotation of the link @ its own axis  $AB$  has no influence on the behaviour of the link.

→  $\boldsymbol{\omega}_n$  and  $\mathbf{r}_{A/B}$  must be at right angles

$$\rightarrow \boldsymbol{\omega}_n \cdot \mathbf{r}_{A/B} = 0$$

→ Additional eqn generated from kinematic requirement

$\mathbf{v}_{A/B}$  is also perpendicular to both  $\boldsymbol{\omega}_n$  and  $\mathbf{r}_{A/B}$   
(since  $\mathbf{v}_A - \mathbf{v}_B = \mathbf{v}_{A/B} = \boldsymbol{\omega}_n \times \mathbf{r}_{A/B}$ )



## Example (1) on general motion

## Velocities of $A$ and $B$ : $v = r\omega$

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega}_n \times \mathbf{r}_{A/B}$$

$$\mathbf{v}_A = 50\omega_2 \mathbf{j} \quad \mathbf{v}_B = 100(6)\mathbf{i} = 600\mathbf{i} \text{ mm/s}$$

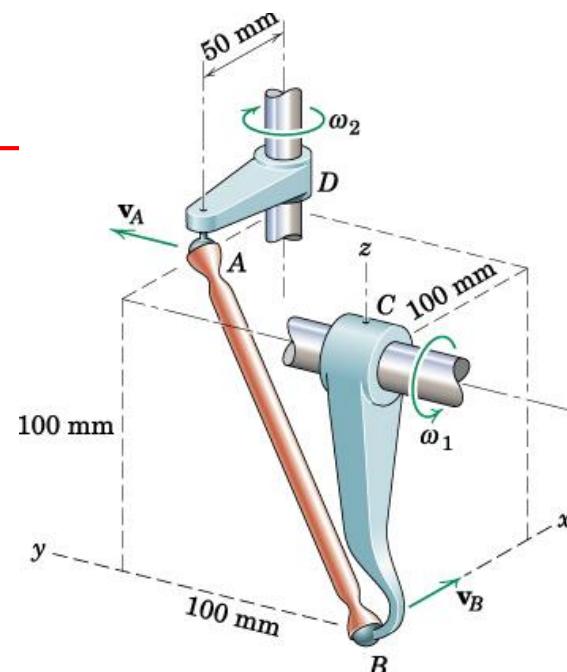
$$\mathbf{r}_{A/B} = 50\mathbf{i} + 100\mathbf{j} + 100\mathbf{k} \text{ mm} \quad (\text{from } B \text{ to } A)$$

$$\rightarrow 50\omega_2\mathbf{j} = 600\mathbf{i} + \begin{vmatrix} \mathbf{i} & \mathbf{j} & \mathbf{k} \\ \omega_{n_x} & \omega_{n_y} & \omega_{n_z} \\ 50 & 100 & 100 \end{vmatrix}$$

## Expanding and equating the coefficients:

$$\begin{aligned} -6 &= \omega_{n_y} - \omega_{n_z} \\ \omega_2 &= -2\omega_{n_x} + \omega_{n_z} \\ 0 &= 2\omega_{n_x} - \omega_{n_y} \end{aligned}$$

Solving:  $\omega_7 = 6 \text{ rad/s}$



$$\omega_n \cdot \mathbf{r}_{A/B} = 0$$

$$50\omega_{px} + 100\omega_{py} + 100\omega_{pz} = 0$$

Substituting in previous two eqns:

$$\omega_{nx} = -4/3 \text{ rad/s}$$

$$\omega_{nv} = -8/3 \text{ rad/s}$$

$$\omega_{nz} = 10/3 \text{ rad/s}$$

$$\omega_n = \frac{2}{3}(-2\mathbf{i} - 4\mathbf{j} + 5\mathbf{k}) \text{ rad/s}$$

$$\omega_n = \frac{2}{3}\sqrt{2^2 + 4^2 + 5^2} = 2\sqrt{5} \text{ rad/s}$$

# Example (1) on general motion

## Angular Acceleration

Linear accln of the links can be found from:

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}}_n \times \mathbf{r}_{A/B} + \boldsymbol{\omega}_n \times (\boldsymbol{\omega}_n \times \mathbf{r}_{A/B})$$

$\boldsymbol{\omega}_n$  :: angular velocity of  $AB$  taken normal to  $AB$

$\dot{\boldsymbol{\omega}}_n$  :: the angular accln of link  $AB$

: Comp of ang vel along  $AB$  has no influence on the motion of the cranks  $C$  and  $D$

.: Contribution of change in magnitude and dirn of this comp of ang vel on actual angular accln of link will not be considered here

→ Only the effect of  $\dot{\boldsymbol{\omega}}_n$

Accln of  $A$  and  $B$  in terms of normal & tangential comp:  $a_n = |\boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r})| = b\omega^2$

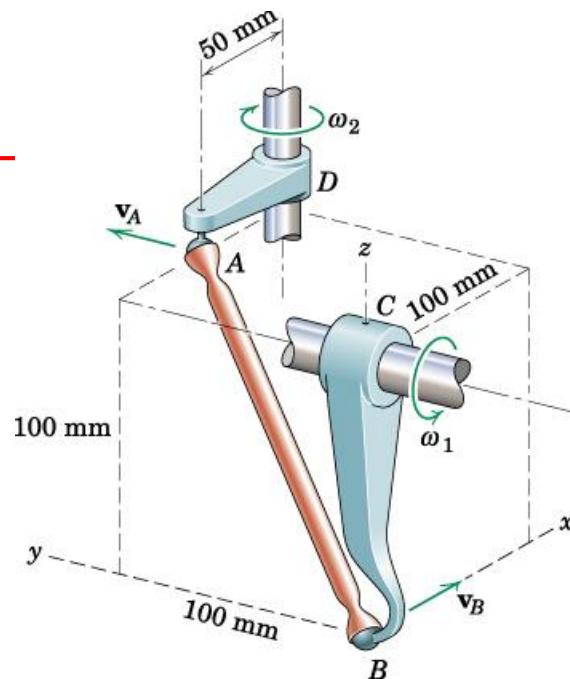
$$\mathbf{a}_A = 50\omega_2^2 \mathbf{i} + 50\dot{\omega}_2 \mathbf{j} = 1800\mathbf{i} + 50\dot{\omega}_2 \mathbf{j} \text{ mm/s}^2$$

$$a_t = |\dot{\boldsymbol{\omega}} \times \mathbf{r}| = b\alpha$$

$$\mathbf{a}_B = 100\omega_1^2 \mathbf{k} + (0)\mathbf{i} = 3600\mathbf{k} \text{ mm/s}^2 \quad \dot{\omega}_1 = 0, \text{ since } \boldsymbol{\omega}_1 \text{ is constant}$$

Further:  $\boldsymbol{\omega}_n \times (\boldsymbol{\omega}_n \times \mathbf{r}_{A/B}) = -\omega_n^2 \mathbf{r}_{A/B} = -20(50\mathbf{i} + 100\mathbf{j} + 100\mathbf{k}) \text{ mm/s}^2$

$$\mathbf{P} \times (\mathbf{Q} \times \mathbf{R}) = (\mathbf{P} \cdot \mathbf{R})\mathbf{Q} - (\mathbf{P} \cdot \mathbf{Q})\mathbf{R} \quad (\text{first term will be zero because } \boldsymbol{\omega}_n \text{ is normal to } \mathbf{r}_{A/B})$$



# Example (1) on general motion

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}}_n \times \mathbf{r}_{A/B} + \boldsymbol{\omega}_n \times (\dot{\boldsymbol{\omega}}_n \times \mathbf{r}_{A/B})$$

$$\begin{aligned}\dot{\boldsymbol{\omega}}_n \times \mathbf{r}_{A/B} &= (100\dot{\omega}_{n_y} - 100\dot{\omega}_{n_z})\mathbf{i} \\ &+ (50\dot{\omega}_{n_z} - 100\dot{\omega}_{n_x})\mathbf{j} + (100\dot{\omega}_{n_x} - 50\dot{\omega}_{n_y})\mathbf{k}\end{aligned}$$

Substituting into rel. accln. eqn and equating respective coefficients:

$$\begin{aligned}28 &= \dot{\omega}_{n_y} - \dot{\omega}_{n_z} \\ \dot{\omega}_2 + 40 &= -2\dot{\omega}_{n_x} + \dot{\omega}_{n_z} \rightarrow \dot{\omega}_2 = -36 \text{ rad/s}^2 \\ -32 &= 2\dot{\omega}_{n_x} - \dot{\omega}_{n_y}\end{aligned}$$

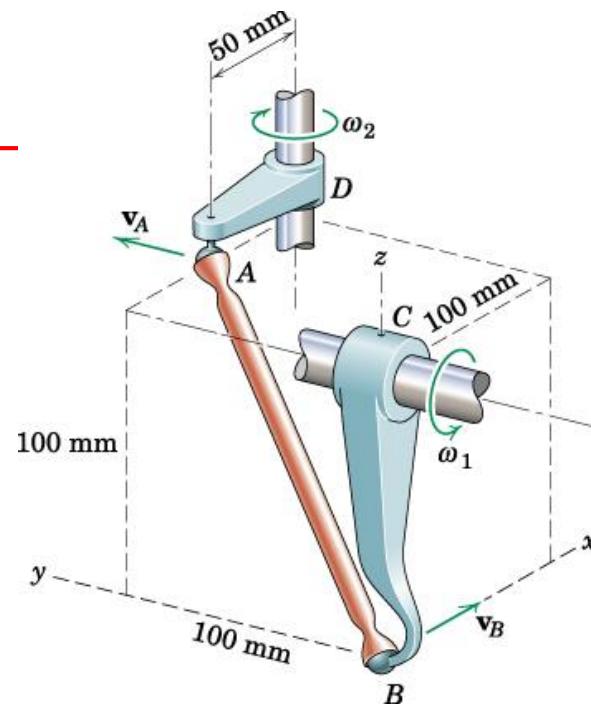
The vector  $\dot{\boldsymbol{\omega}}_n$  is normal to  $\mathbf{r}_{A/B}$  (but is not normal to  $\mathbf{v}_{A/B}$ ). The component of  $\dot{\boldsymbol{\omega}}_n$  which is not normal to  $\mathbf{v}_{A/B}$  gives rise to the change in direction of  $\mathbf{v}_{A/B}$ .

$$[\dot{\boldsymbol{\omega}}_n \cdot \mathbf{r}_{A/B} = 0] \quad 2\dot{\omega}_{n_x} + 4\dot{\omega}_{n_y} + 4\dot{\omega}_{n_z} = 0$$

Substituting in previous two eqns:  $\dot{\omega}_{n_x} = -8 \text{ rad/s}^2$   $\dot{\omega}_{n_y} = 16 \text{ rad/s}^2$   $\dot{\omega}_{n_z} = -12 \text{ rad/s}^2$

$$\dot{\boldsymbol{\omega}}_n = 4(-2\mathbf{i} + 4\mathbf{j} - 3\mathbf{k}) \text{ rad/s}^2$$

$$|\dot{\boldsymbol{\omega}}_n| = 4\sqrt{2^2 + 4^2 + 3^2} = 4\sqrt{29} \text{ rad/s}^2$$



# 3-D Kinematics

## Rotating Reference Axes

: Reference axes translate as well as rotate

: Origin of the reference axes attached to  $B$

The reference axes x-y-z rotate with an absolute angular velocity  $\Omega$ , which may be different from absolute angular velocity  $\omega$  of the body

For plane motion:

$$\dot{\mathbf{i}} = \boldsymbol{\omega} \times \mathbf{i} \quad \text{and} \quad \dot{\mathbf{j}} = \boldsymbol{\omega} \times \mathbf{j}$$

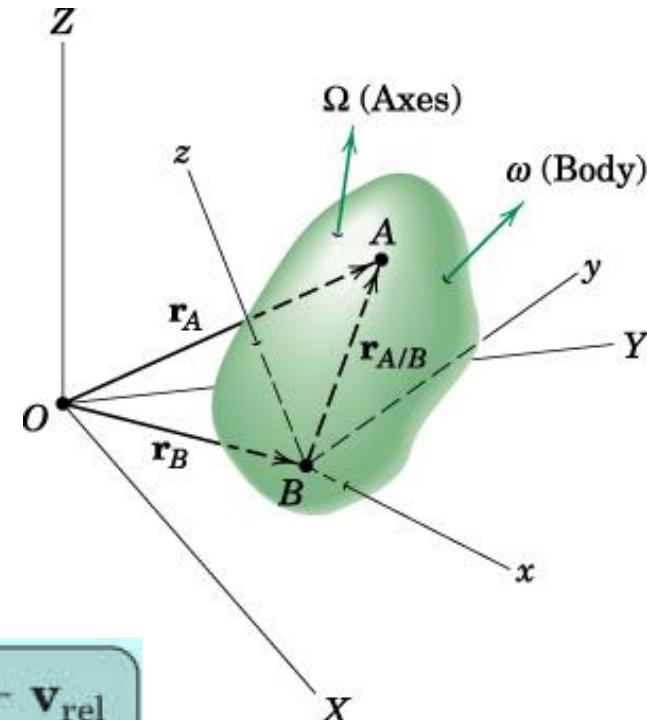
$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega} \times \mathbf{r} + \mathbf{v}_{\text{rel}}$$

$$\left( \frac{d\mathbf{V}}{dt} \right)_{XY} = \left( \frac{d\mathbf{V}}{dt} \right)_{xy} + \boldsymbol{\omega} \times \mathbf{V}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

For 3-D motion:

- include the z-component of the vectors



# 3-D Kinematics

## Rotating Reference Axes

$$\dot{\mathbf{i}} = \boldsymbol{\omega} \times \mathbf{i} \quad \text{and} \quad \dot{\mathbf{j}} = \boldsymbol{\omega} \times \mathbf{j}$$

$$\dot{\mathbf{i}} = \boldsymbol{\Omega} \times \mathbf{i} \quad \dot{\mathbf{j}} = \boldsymbol{\Omega} \times \mathbf{j} \quad \dot{\mathbf{k}} = \boldsymbol{\Omega} \times \mathbf{k}$$

→ Time derivatives of the unit vectors attached to  $x$ - $y$ - $z$

Vel and accln of point A:

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\omega} \times \mathbf{r} + \mathbf{v}_{\text{rel}}$$

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\Omega} \times \mathbf{r}_{A/B} + \mathbf{v}_{\text{rel}}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\Omega}} \times \mathbf{r}_{A/B} + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{r}_{A/B}) + 2\boldsymbol{\Omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

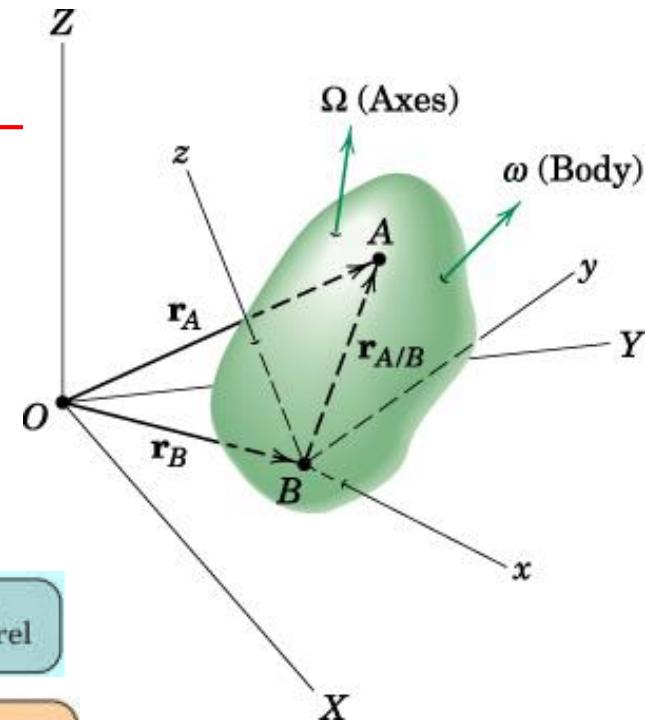
Vel and accln of point A measured rel. to  $x$ - $y$ - $z$  by an observer attached to  $x$ - $y$ - $z$

$$\mathbf{v}_{\text{rel}} = \dot{x}\mathbf{i} + \dot{y}\mathbf{j} + \dot{z}\mathbf{k} \text{ and } \mathbf{a}_{\text{rel}} = \ddot{x}\mathbf{i} + \ddot{y}\mathbf{j} + \ddot{z}\mathbf{k}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\omega}} \times \mathbf{r} + \boldsymbol{\omega} \times (\boldsymbol{\omega} \times \mathbf{r}) + 2\boldsymbol{\omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

: If  $x$ - $y$ - $z$  are rigidly attached to the body,

→  $\boldsymbol{\Omega} = \boldsymbol{\omega}$ , and  $\mathbf{v}_{\text{rel}} = \mathbf{a}_{\text{rel}} = 0$  → motion will be the same as for translating reference axes



For different  $\boldsymbol{\Omega}$  and  $\boldsymbol{\omega}$ , the term  $\mathbf{r}_{A/B}$  remains **constant in magnitude** but will **change its direction** wrt  $x$ - $y$ - $z$

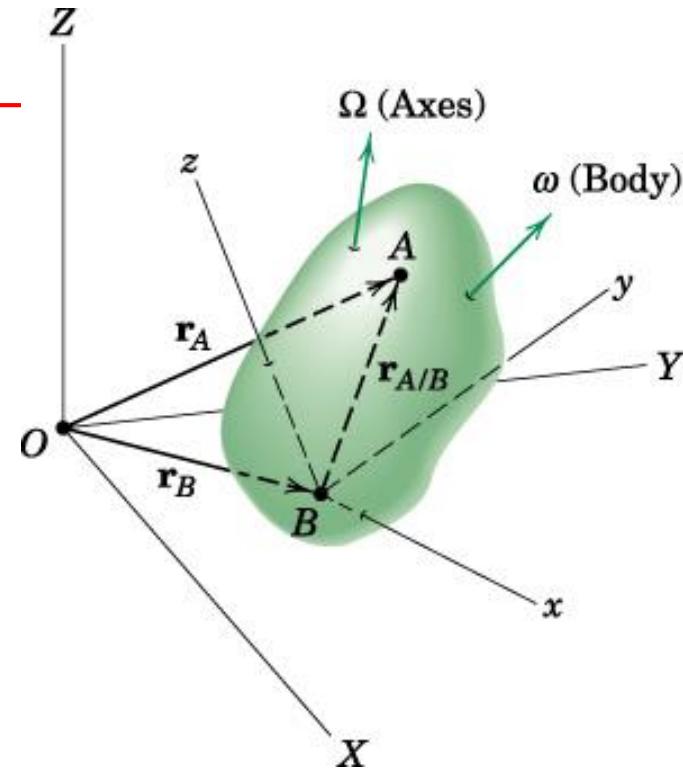
# 3-D Kinematics

## Rotating Reference Axes

$$\left(\frac{d\mathbf{V}}{dt}\right)_{XY} = \left(\frac{d\mathbf{V}}{dt}\right)_{xy} + \boldsymbol{\omega} \times \mathbf{V} \quad \text{for any vector}$$

Eqn relating time derivative of a vector  $\mathbf{V}$  measured in fixed  $X$ - $Y$ - $Z$  system and the time derivative of  $\mathbf{V}$  measured relative to rotating  $x$ - $y$ - $z$  system:

$$\left(\frac{d\mathbf{V}}{dt}\right)_{XYZ} = \left(\frac{d\mathbf{V}}{dt}\right)_{xyz} + \boldsymbol{\Omega} \times \mathbf{V}$$



Applying this transformation to the relative posn vector:

$$\mathbf{r}_{A/B} = \mathbf{r}_A - \mathbf{r}_B \quad \rightarrow \quad \left(\frac{d\mathbf{r}_A}{dt}\right)_{XYZ} = \left(\frac{d\mathbf{r}_B}{dt}\right)_{XYZ} + \left(\frac{d\mathbf{r}_{A/B}}{dt}\right)_{xyz} + \boldsymbol{\Omega} \times \mathbf{r}_{A/B}$$

→ The same eqn is obtained

$$\mathbf{v}_A = \mathbf{v}_B + \mathbf{v}_{\text{rel}} + \boldsymbol{\Omega} \times \mathbf{r}_{A/B}$$

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\Omega} \times \mathbf{r}_{A/B} + \mathbf{v}_{\text{rel}}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\Omega}} \times \mathbf{r}_{A/B} + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{r}_{A/B}) + 2\boldsymbol{\Omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

# 3-D Kinematics

## Rotating Reference Axes

$$\left(\frac{d\mathbf{V}}{dt}\right)_{XYZ} = \left(\frac{d\mathbf{V}}{dt}\right)_{xyz} + \boldsymbol{\Omega} \times \mathbf{V}$$

This eqn may be recast as the vector operator for general use:

$$\left(\frac{d[\ ]}{dt}\right)_{XYZ} = \left(\frac{d[\ ]}{dt}\right)_{xyz} + \boldsymbol{\Omega} \times [ ]$$

[ ] stands for any vector  $\mathbf{V}$  expressible both in  $X$ - $Y$ - $Z$

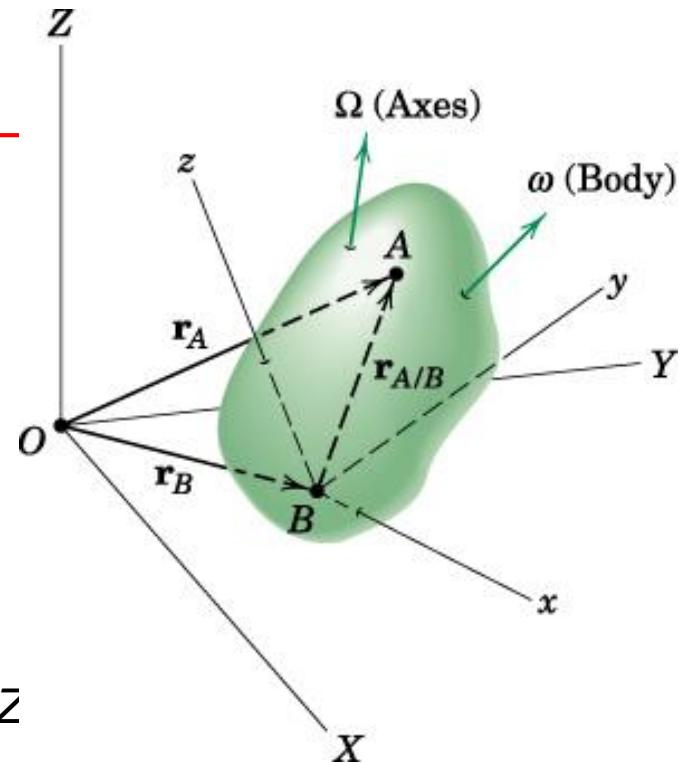
Applying the operator to itself  $\rightarrow$  second time derivative.

$$\left(\frac{d^2[\ ]}{dt^2}\right)_{XYZ} = \left(\frac{d^2[\ ]}{dt^2}\right)_{xyz} + \dot{\boldsymbol{\Omega}} \times [ ] + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times [ ]) + 2\boldsymbol{\Omega} \times \left(\frac{d[\ ]}{dt}\right)_{xyz}$$

$\rightarrow$  The same eqn is for accln

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\Omega} \times \mathbf{r}_{A/B} + \mathbf{v}_{\text{rel}}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\Omega}} \times \mathbf{r}_{A/B} + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{r}_{A/B}) + 2\boldsymbol{\Omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$



# Example on Rotating Reference Axes

The motor housing and its bracket rotate about the Z-axis at the constant rate  $\Omega = 3 \text{ rad/s}$ . The motor shaft and disk have a constant angular velocity of spin  $p = 8 \text{ rad/s}$  with respect to the motor housing in the direction shown. If  $\gamma$  is constant at  $30^\circ$ , determine the velocity and acceleration of point A at the top of the disk and the angular acceleration  $\alpha$  of the disk.

## Solution

The rotating reference axes x-y-z are attached to the motor housing.

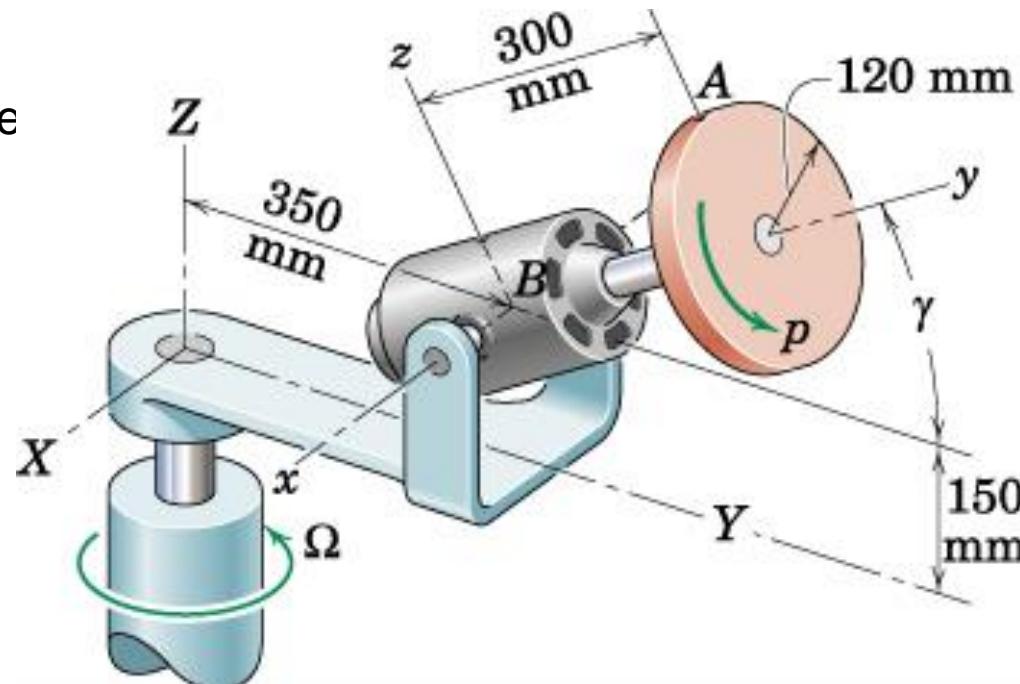
Unit vectors for X-Y-Z:  $\mathbf{i}, \mathbf{j}, \mathbf{k}$   
Unit vectors for x-y-z:  $\mathbf{i}, \mathbf{j}, \mathbf{k}$

Angular vel of x-y-z:

$$\boldsymbol{\Omega} = \Omega \mathbf{k} = 3 \mathbf{k} \text{ rad/s}$$

$$\mathbf{v}_A = \mathbf{v}_B + \boldsymbol{\Omega} \times \mathbf{r}_{A/B} + \mathbf{v}_{\text{rel}}$$

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\boldsymbol{\Omega}} \times \mathbf{r}_{A/B} + \boldsymbol{\Omega} \times (\boldsymbol{\Omega} \times \mathbf{r}_{A/B}) + 2\boldsymbol{\Omega} \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$



# Example on Rotating Reference Axes

Velocity of A:

$$\Omega = \Omega \mathbf{K} = 3 \mathbf{K} \text{ rad/s}$$

$$\mathbf{v}_A = \mathbf{v}_B + \Omega \times \mathbf{r}_{A/B} + \mathbf{v}_{\text{rel}}$$

$$\mathbf{v}_B = \Omega \times \mathbf{r}_B = 3 \mathbf{K} \times 0.350 \mathbf{J} = -1.05 \mathbf{I} = -1.05 \mathbf{i} \text{ m/s}$$

In  $\mathbf{r}_B$ ,  $\mathbf{K}$  component is not considered  
because  $\mathbf{K} \times \mathbf{K} = 0$

$$\Omega \times \mathbf{r}_{A/B} = 3 \mathbf{K} \times (0.300 \mathbf{j} + 0.120 \mathbf{k})$$

$$\mathbf{K} \times \mathbf{i} = \mathbf{J} = \mathbf{j} \cos \gamma - \mathbf{k} \sin \gamma$$

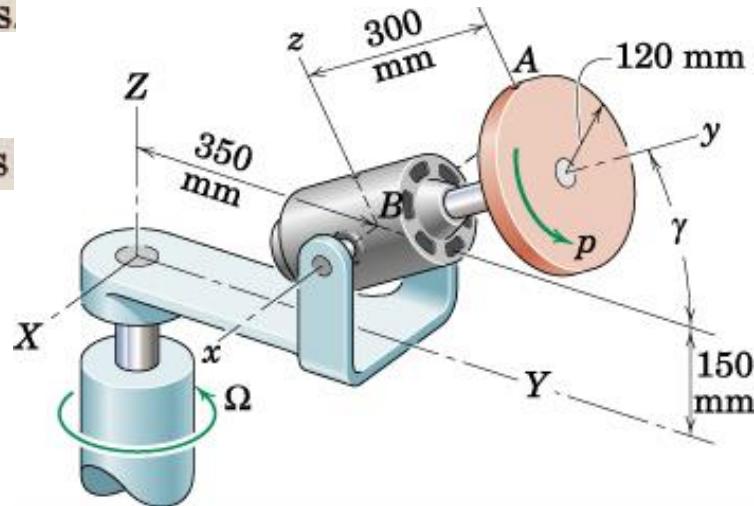
$$\mathbf{K} \times \mathbf{j} = -\mathbf{i} \cos \gamma$$

$$\mathbf{K} \times \mathbf{k} = \mathbf{i} \sin \gamma$$

$$\Omega \times \mathbf{r}_{A/B} = (-0.9 \cos 30^\circ) \mathbf{i} + (0.36 \sin 30^\circ) \mathbf{i} = -0.599 \mathbf{i} \text{ m/s}$$

$$\mathbf{v}_{\text{rel}} = \mathbf{p} \times \mathbf{r}_{A/B} = 8 \mathbf{j} \times (0.300 \mathbf{j} + 0.120 \mathbf{k}) = 0.960 \mathbf{i} \text{ m/s}$$

$$\rightarrow \mathbf{v}_A = -1.05 \mathbf{i} - 0.599 \mathbf{i} + 0.960 \mathbf{i} = -0.689 \mathbf{i} \text{ m/s}$$



# Example on Rotating Reference Axes

## Acceleration of A

$$\mathbf{a}_A = \mathbf{a}_B + \dot{\Omega} \times \mathbf{r}_{A/B} + \Omega \times (\Omega \times \mathbf{r}_{A/B}) + 2\Omega \times \mathbf{v}_{\text{rel}} + \mathbf{a}_{\text{rel}}$$

Here:

$$\begin{aligned}\mathbf{a}_B &= \Omega \times (\Omega \times \mathbf{r}_B) = 3\mathbf{K} \times (3\mathbf{K} \times 0.350\mathbf{J}) = -3.15\mathbf{J} \\ &= 3.15(-\mathbf{j} \cos 30^\circ + \mathbf{k} \sin 30^\circ) = -2.73\mathbf{j} + 1.575\mathbf{k} \text{ m/s}^2\end{aligned}$$

$$\dot{\Omega} = \mathbf{0}$$

$$\begin{aligned}\Omega \times (\Omega \times \mathbf{r}_{A/B}) &= 3\mathbf{K} \times [3\mathbf{K} \times (0.300\mathbf{j} + 0.120\mathbf{k})] \\ &= 3\mathbf{K} \times (-0.599\mathbf{i}) = -1.557\mathbf{j} + 0.899\mathbf{k} \text{ m/s}^2\end{aligned}$$

$$\begin{aligned}2\Omega \times \mathbf{v}_{\text{rel}} &= 2(3\mathbf{K}) \times 0.960\mathbf{i} = 5.76\mathbf{J} \\ &= 5.76(\mathbf{j} \cos 30^\circ - \mathbf{k} \sin 30^\circ) = 4.99\mathbf{j} - 2.88\mathbf{k} \text{ m/s}^2\end{aligned}$$

$$\mathbf{a}_{\text{rel}} = \mathbf{p} \times (\mathbf{p} \times \mathbf{r}_{A/B}) = 8\mathbf{j} \times [8\mathbf{j} \times (0.300\mathbf{j} + 0.120\mathbf{k})] = -7.68\mathbf{k} \text{ m/s}^2$$

$$\rightarrow \mathbf{a}_A = 0.703\mathbf{j} - 8.09\mathbf{k} \text{ m/s}^2 \quad a_A = \sqrt{(0.703)^2 + (8.09)^2} = 8.12 \text{ m/s}^2$$

## Angular Accln:

$$\alpha = \dot{\omega} = \Omega \times \omega = 3\mathbf{K} \times (3\mathbf{K} + 8\mathbf{j}) = \mathbf{0} + (-24 \cos 30^\circ)\mathbf{i} = -20.8\mathbf{i} \text{ rad/s}^2$$

